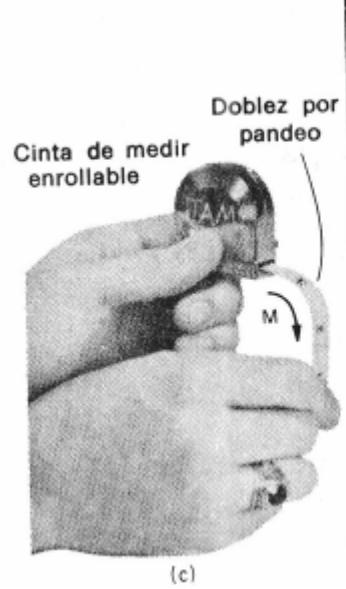
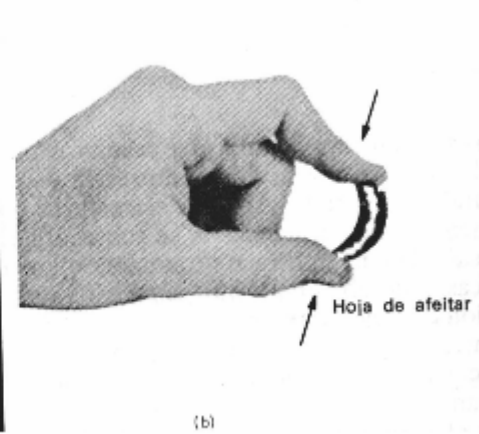
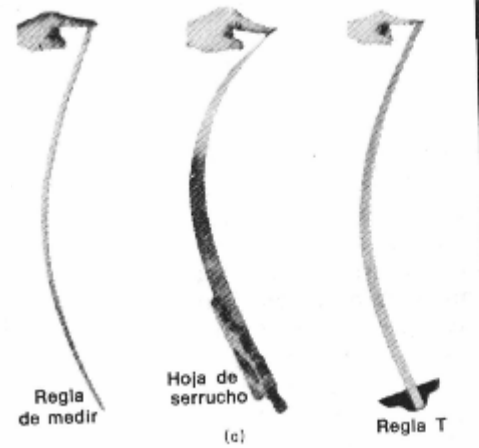
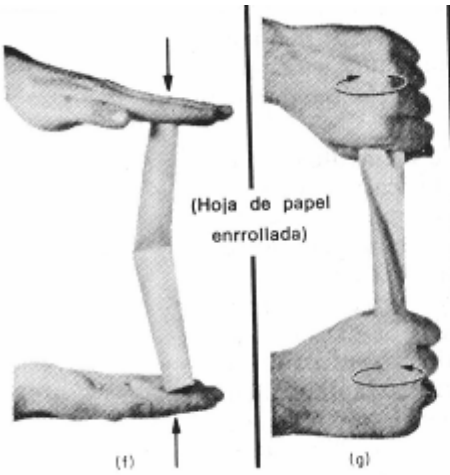

Pandeo

Clase 20

Pandeo Elástico – Pandeo Inelástico

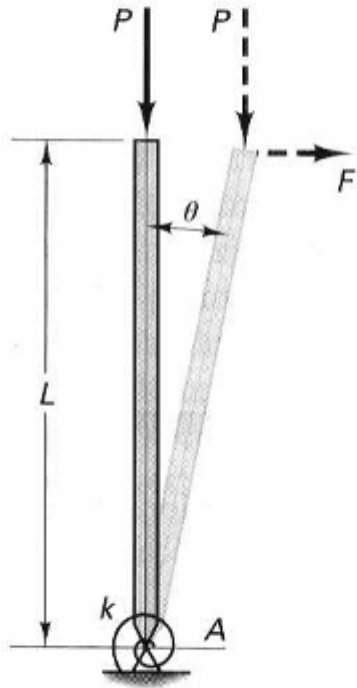




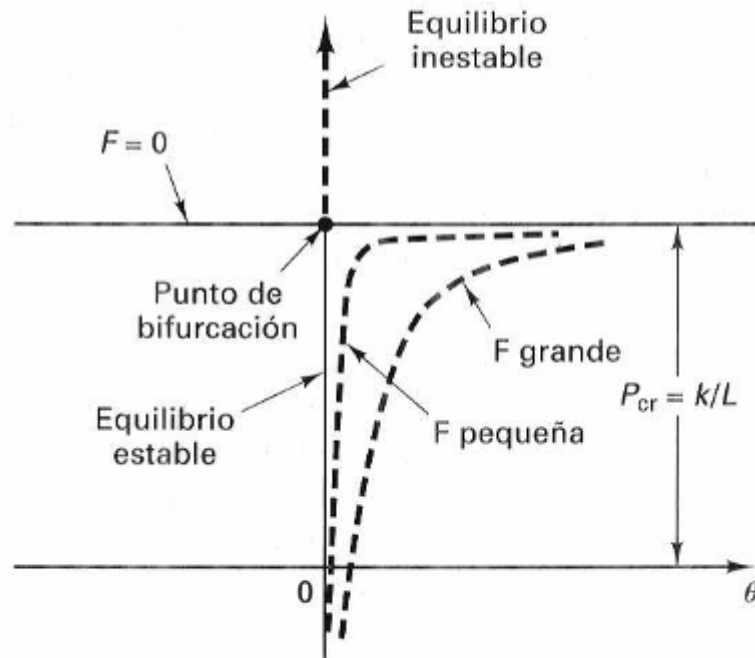
$k\theta > PL\theta$ el sistema es *estable*

$k\theta < PL\theta$ el sistema es *inestable*

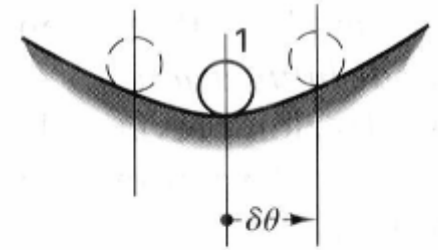
y



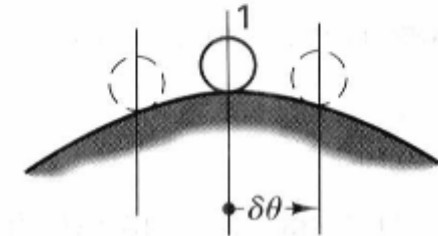
(a)



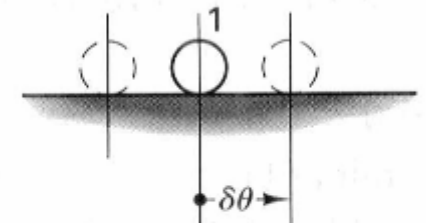
(b)



(a)



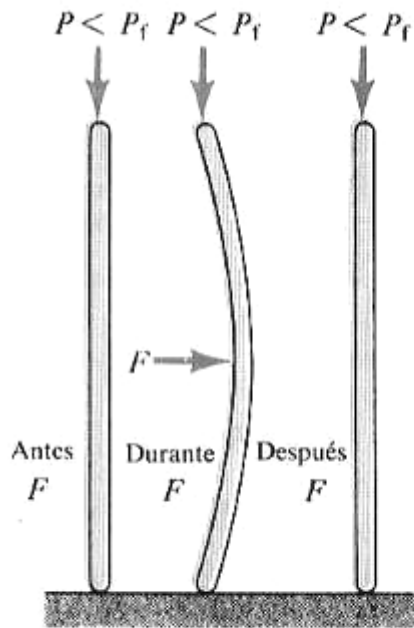
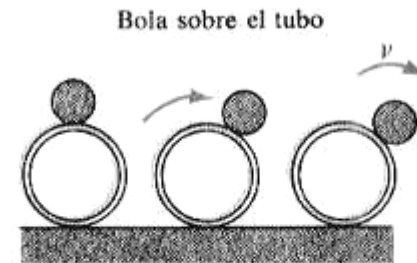
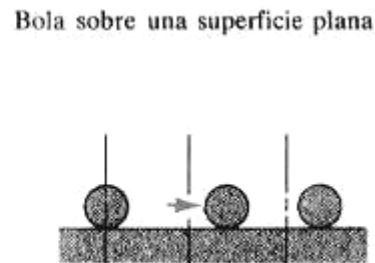
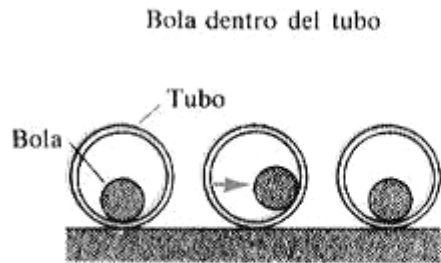
(b)



(c)

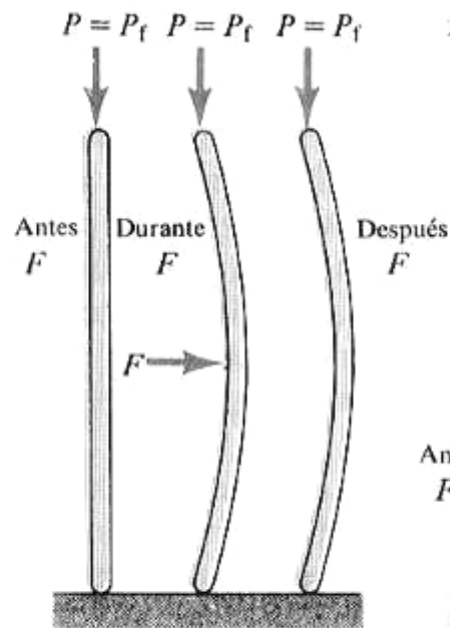
Equilibrios (a) estable, (b) inestable y (c) neutro.

Comportamiento por pandeo de una barra rígida.



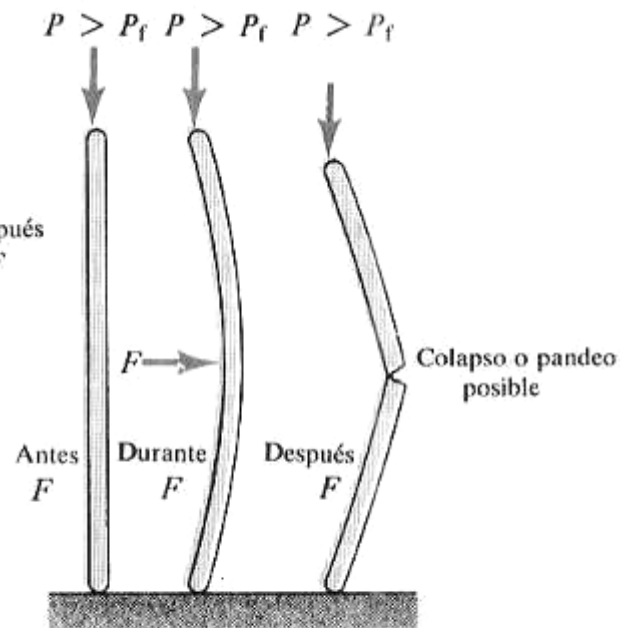
Equilibrio estable

(a)



Equilibrio precario

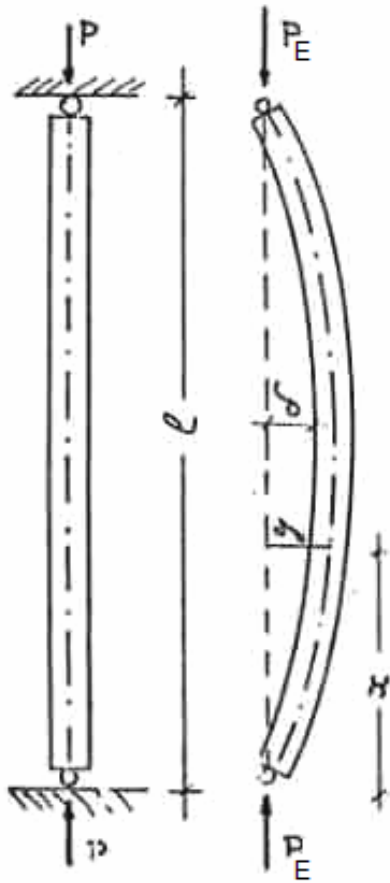
(b)



Equilibrio inestable

(c)

FORMULA DE EULER



$$EI \frac{d^2 y}{dx^2} = -Py \quad \text{SI} \quad k^2 = \frac{P}{EI}$$

$$\frac{d^2 y}{dx^2} + k^2 y = 0$$

$$y = C_1 \sin kx + C_2 \cos kx$$

$$y = 0 \quad \text{para} \quad x = 0 \quad ; \quad x = l$$

$$C_2 = 0$$

$$0 = C_1 \sin kl$$

$$\underline{kl = n\pi} \quad n = 0, 1, 2, \dots$$

PARA $n = 1$

$$P_E = P_{crit} = \frac{\pi^2 EI}{l^2}$$

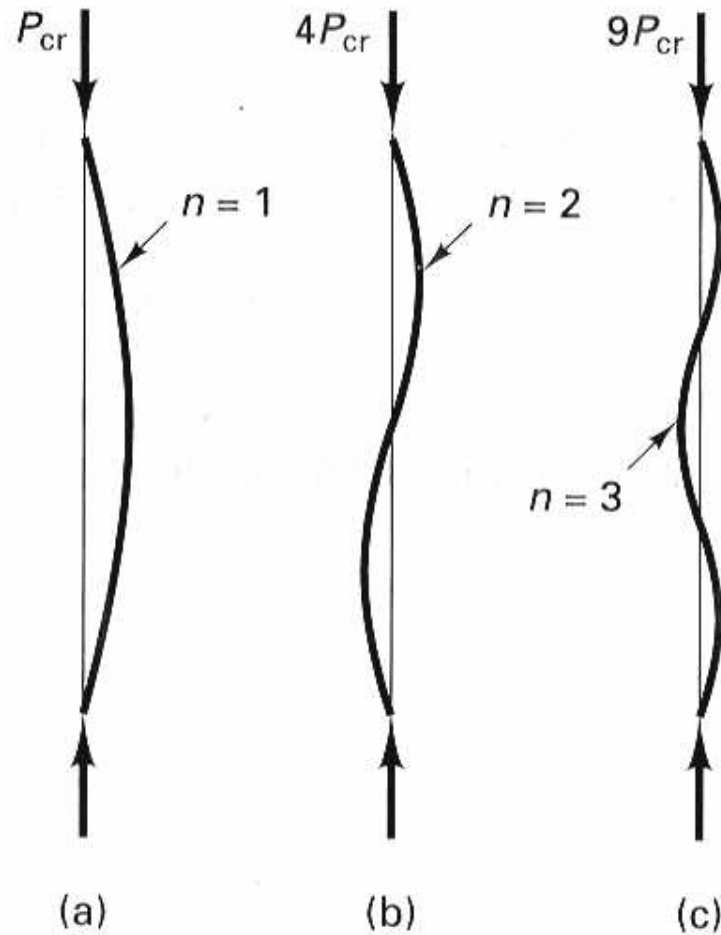
CASOS TEÓRICOS

para $x = \frac{l}{2}$ $y = \delta$

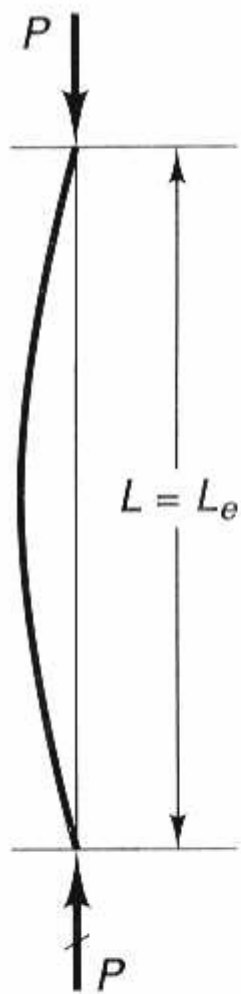
$$\delta = C_1 \operatorname{sen} \frac{n\pi}{l} \frac{l}{2}$$

$$\delta = C_1$$

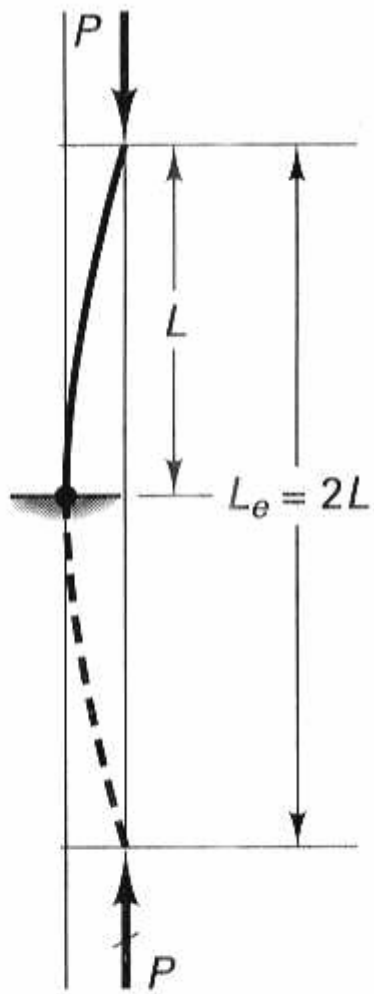
$$y = \delta \operatorname{sen} \frac{n\pi}{l} x$$



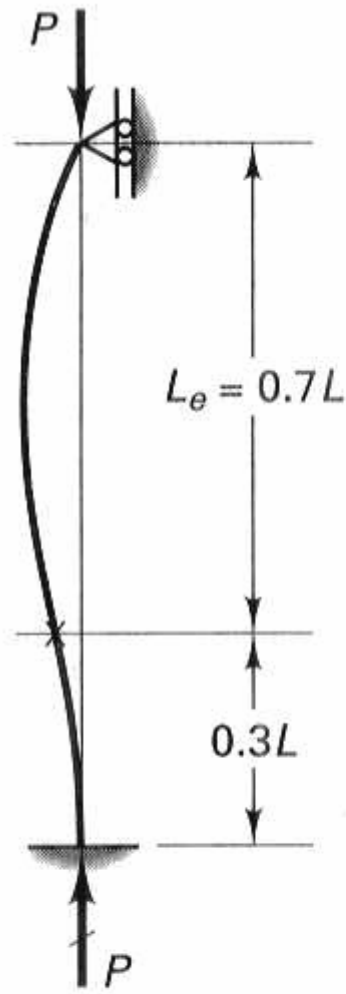
Tres primeros modos de pandeo para una columna articulada en ambos extremos.



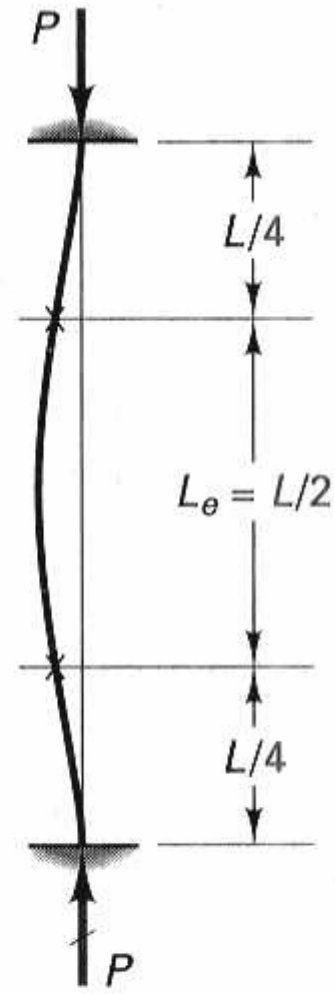
(a)



(b)



(c)



(d)

Longitudes efectivas de columnas con diferentes restricciones.

ESBELTEZ GEOMETRICA :

$$\frac{b}{a}$$

a = menor dimension

ESBELTEZ MECANICA :

$$\lambda = \frac{l_p}{i_{\min}}$$

i = radio de giro mínimo

CURVA DE EULER

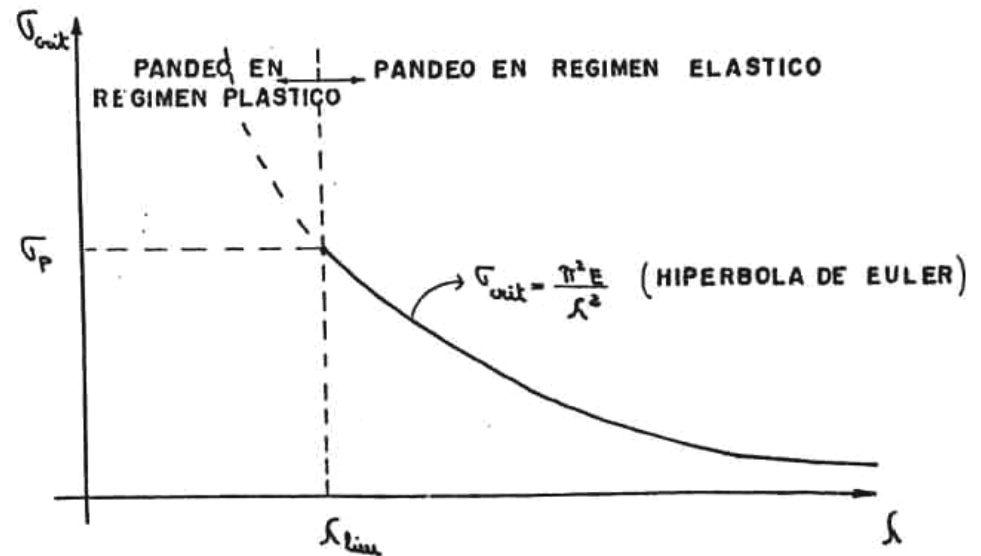
$$P_{\text{crit}} = \frac{\pi^2 EI}{l_p^2} = \pi^2 EA \left(\frac{l}{l_p}\right)^2 = \pi^2 \frac{EA}{\lambda^2}$$

$$\sigma_{\text{crit}} = \frac{\pi^2 E}{\lambda^2}$$

$$\sigma_{\text{crit}} = \frac{\pi^2 E}{\lambda^2}$$

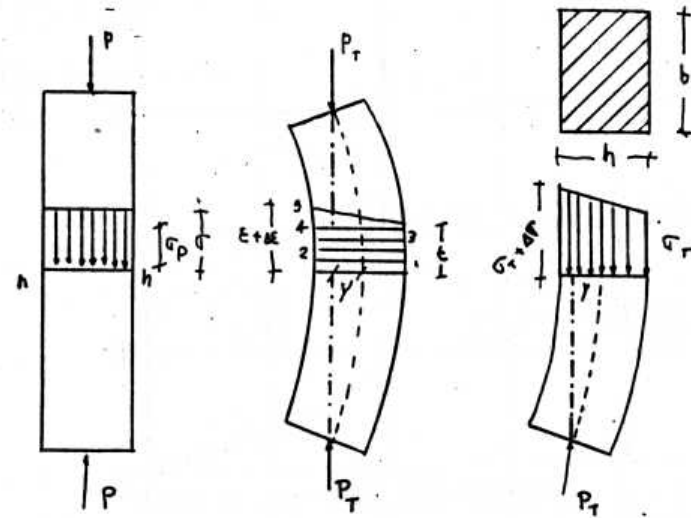
$$\sigma_p = \frac{\pi^2 E}{\lambda_{\text{lim}}^2}$$

$$\lambda_{\text{lim}} = \pi \sqrt{\frac{E}{\sigma_p}}$$

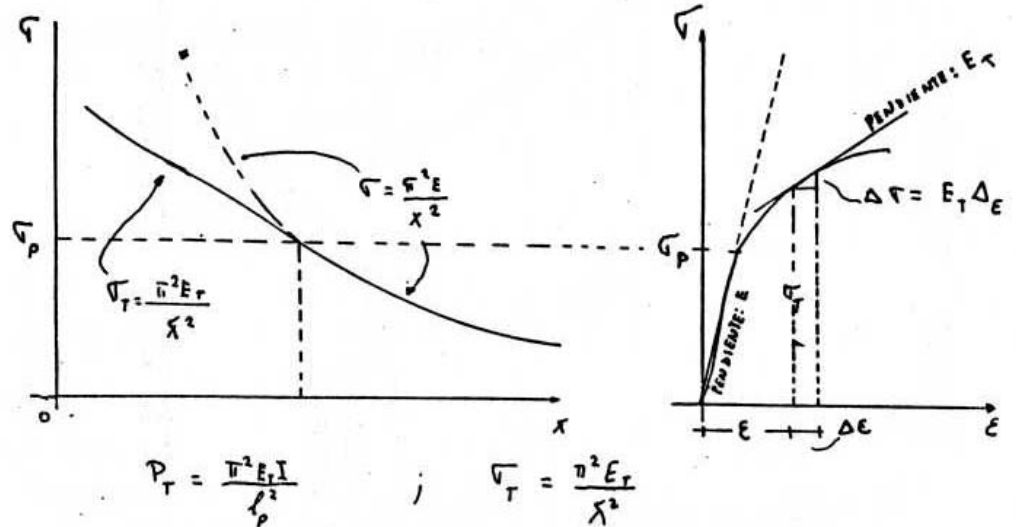


PANDEO INELÁSTICO

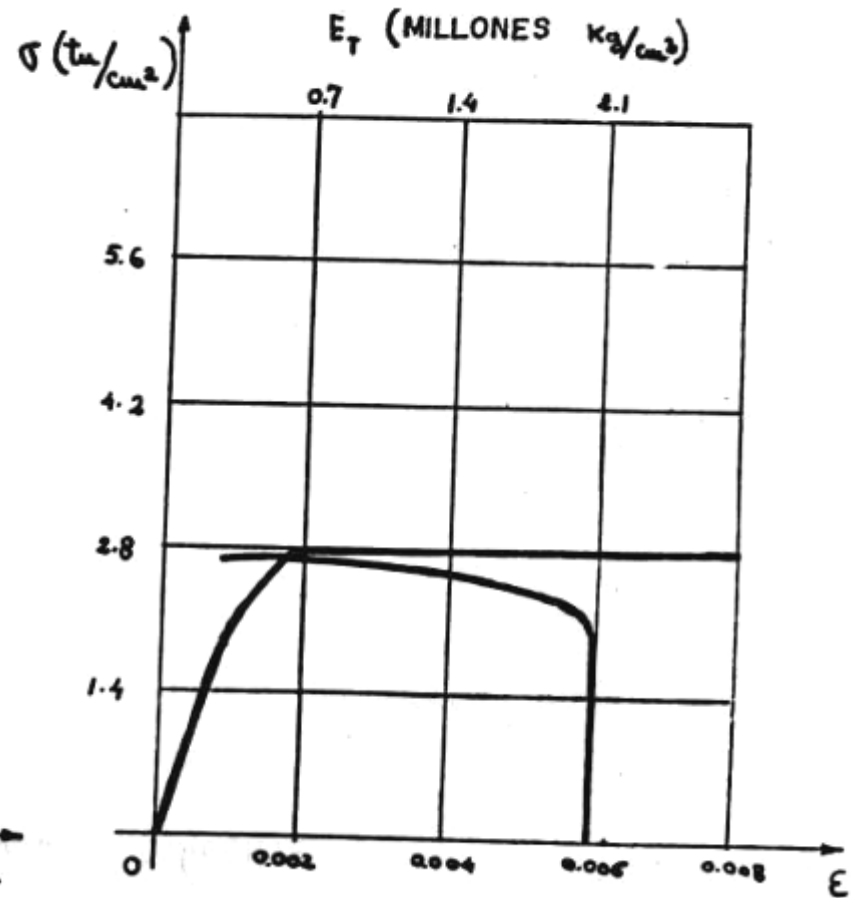
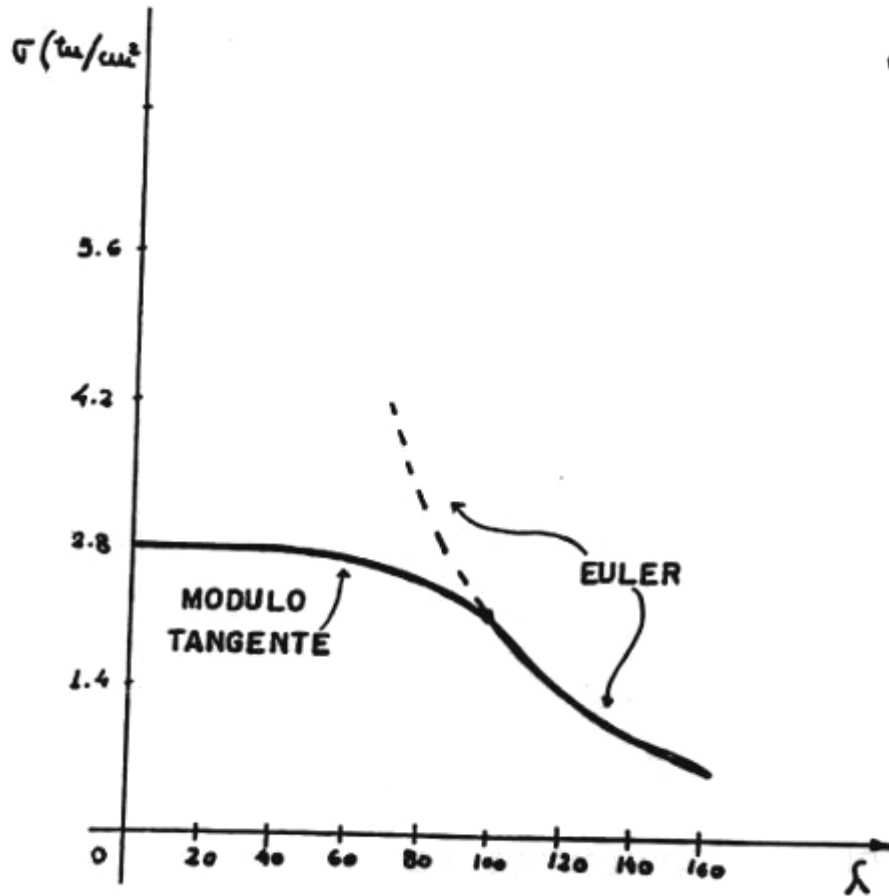
FORMULA DEL MODULO TANGENCIAL



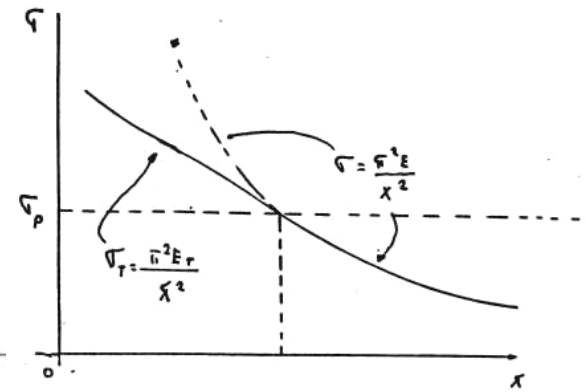
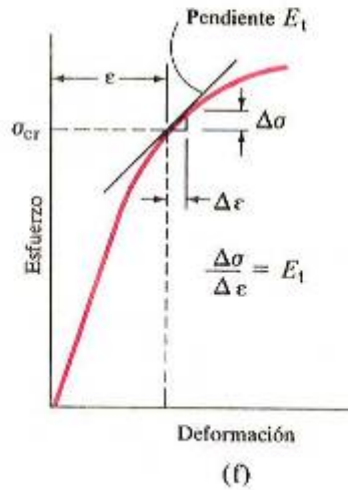
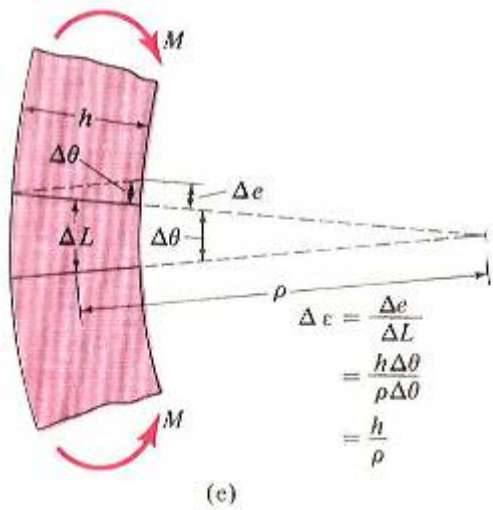
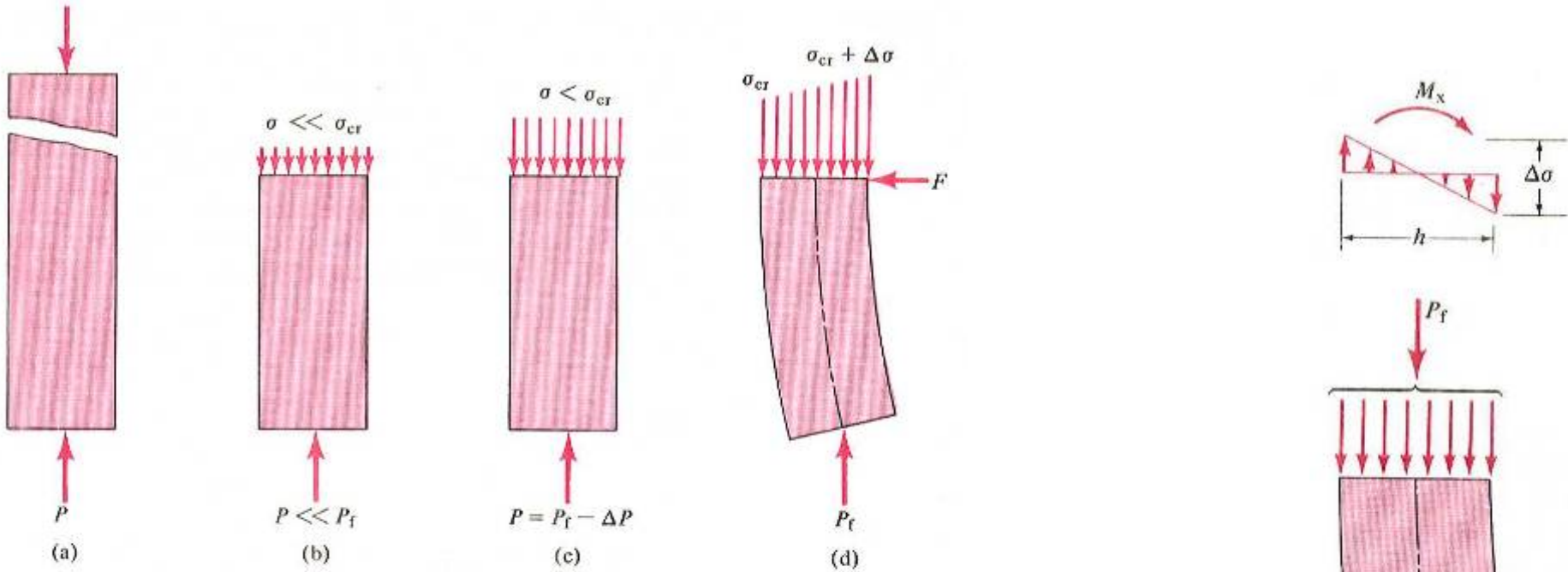
$$P_T \cdot y = \frac{\Delta \tau \cdot I}{\frac{h}{2}} \quad ; \quad P_T \cdot y = \frac{E_T \Delta \epsilon \cdot I}{h} \quad ; \quad P_T \cdot y = \frac{E_T I}{R} \quad ; \quad E_T I \frac{d^2 y}{dx^2} = - P_T y$$



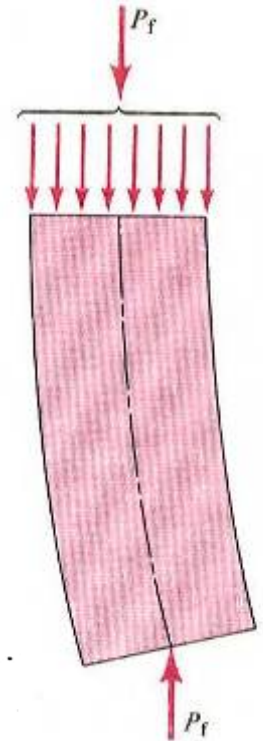
PANDEO INELÁSTICO – MODULO TANGENTE



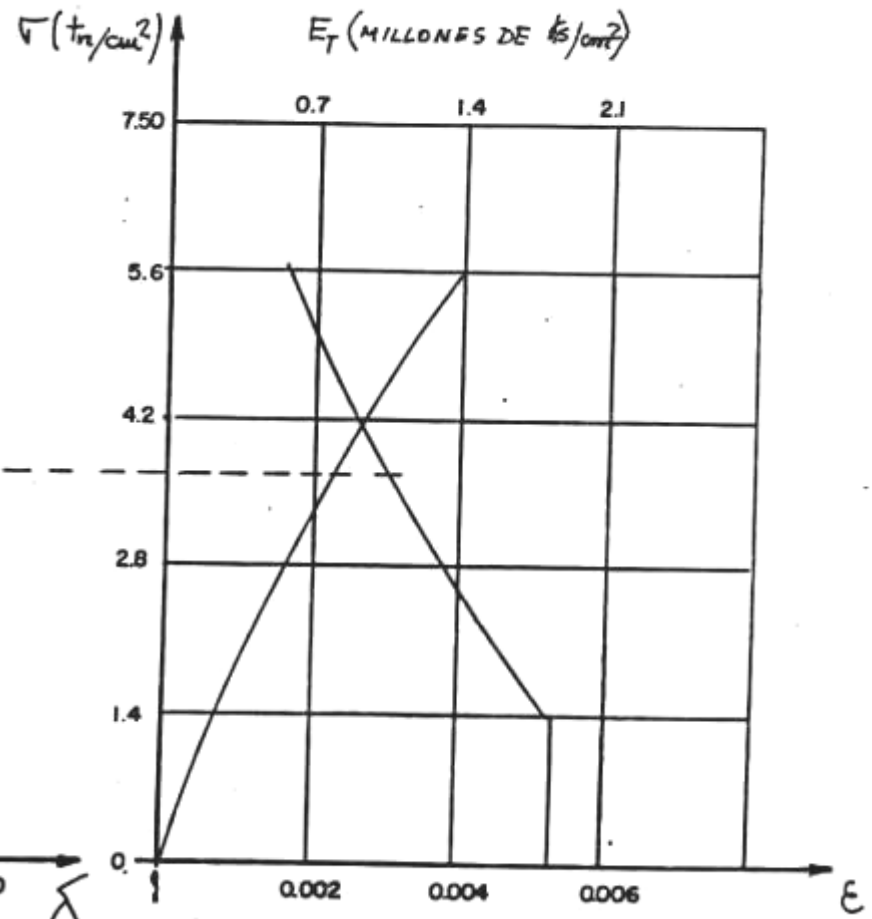
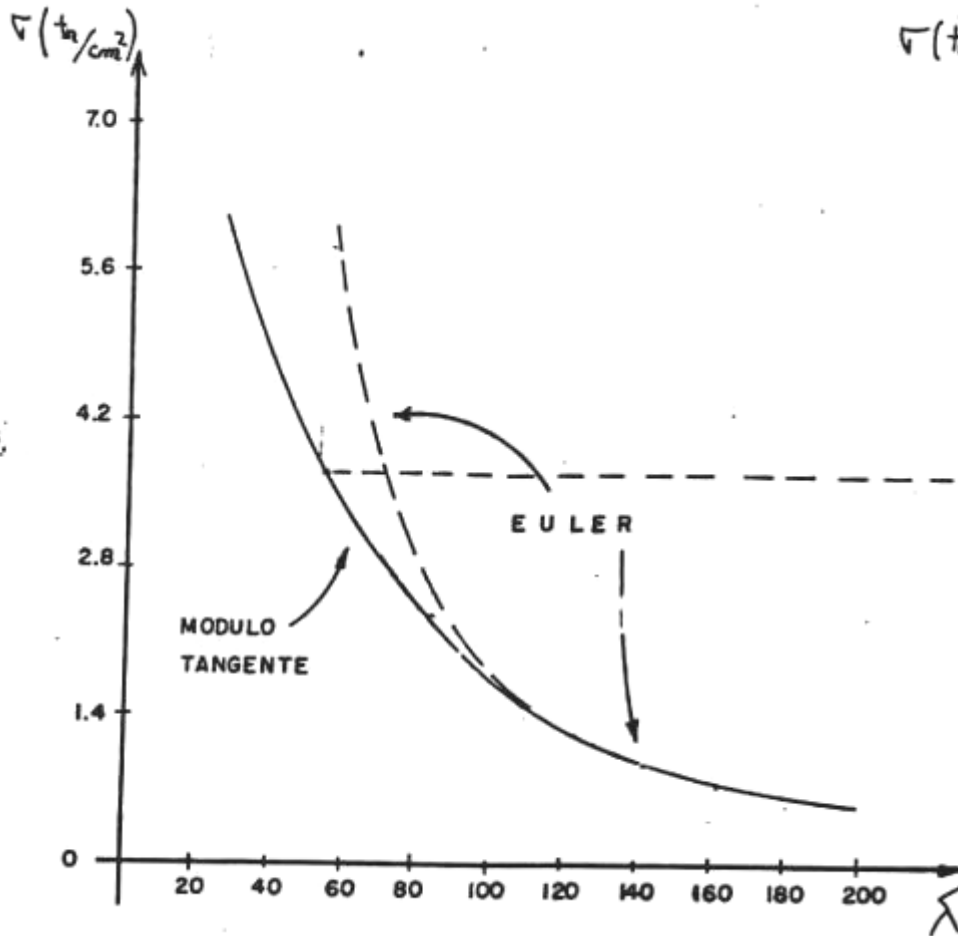
PANDEO INELÁSTICO – FORMULA DEL DOBLE MÓDULO TANGENCIAL



$$P_T = \frac{P^2 E_f I}{\chi^2} \quad ; \quad V_T = \frac{P^2 E_f}{\chi^2}$$

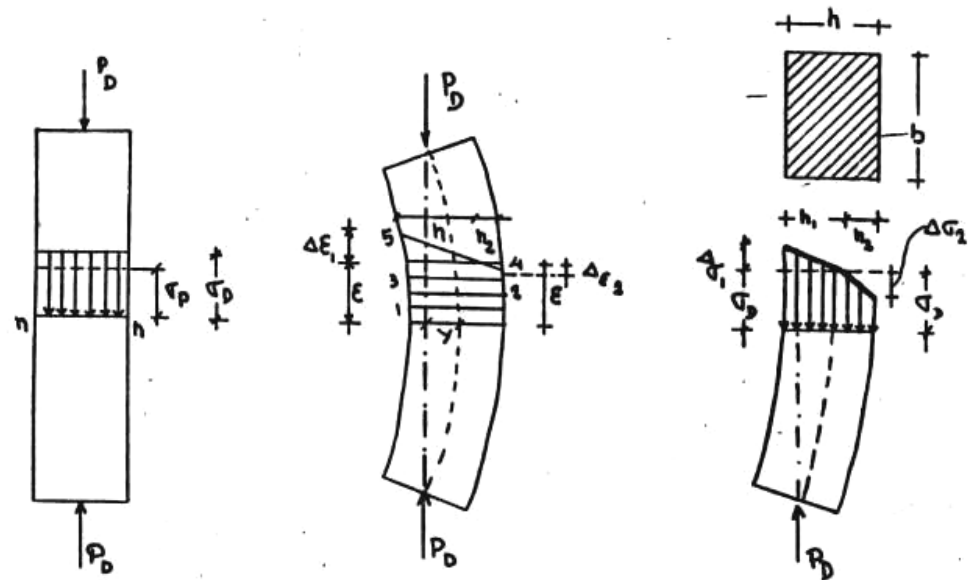


PANDEO INELASTICO – MODULO TANGENTE



Acero inoxidable

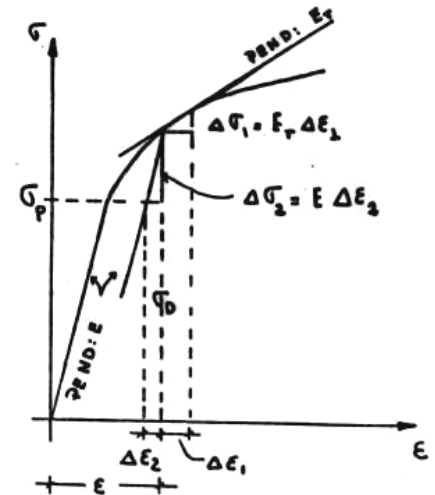
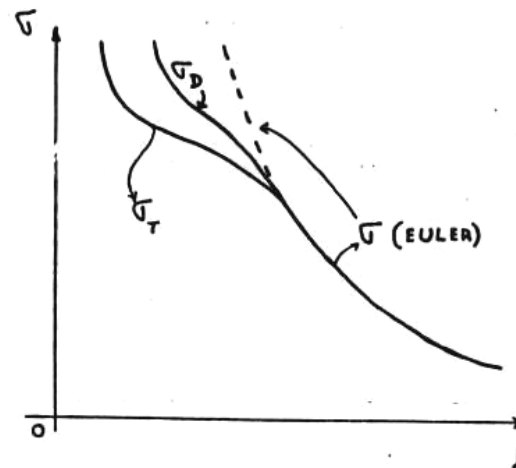
PANDEO INELASTICO: TEORÍA DEL DOBLE MÓDULO



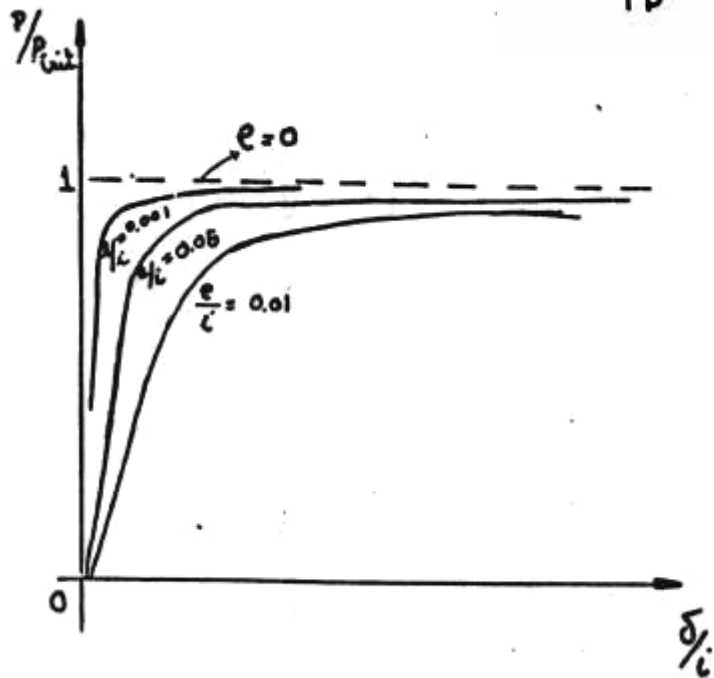
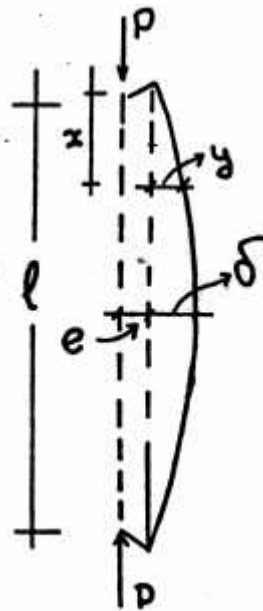
$$P_D = \frac{\pi^2 E_D I}{\lambda^2}$$

$$P_D = \frac{\pi^2 E_D}{\lambda^2}$$

$$E_D = \frac{4 E_T}{\left(1 + \sqrt{\frac{E_T}{E}}\right)^2}$$



FORMULA DE LA SECANTE O DE SCHEFFLER



$$M = P(e + y)$$

$$k^2 = \frac{P}{EI}$$

$$\frac{d^2 y}{dx^2} + k^2 y = -k^2 e$$

$$y = e \left(\tan \frac{kl}{2} \sec kx + \cos kx - 1 \right)$$

$$\delta = e \left(\sec \frac{kl}{2} - 1 \right)$$

$$\sigma_{max} = \frac{P}{A} \left[1 + \frac{e \cdot c}{i^2} \sec \left(\frac{kl}{2} \right) \right]$$

$$\sigma_{max} = \frac{P}{A} \left[1 + \frac{e \cdot c}{i^2} \sec \left(\frac{l}{2i} \sqrt{\frac{P}{AE}} \right) \right]$$

$$M_{max} = Pe \sec \frac{kl}{2}$$

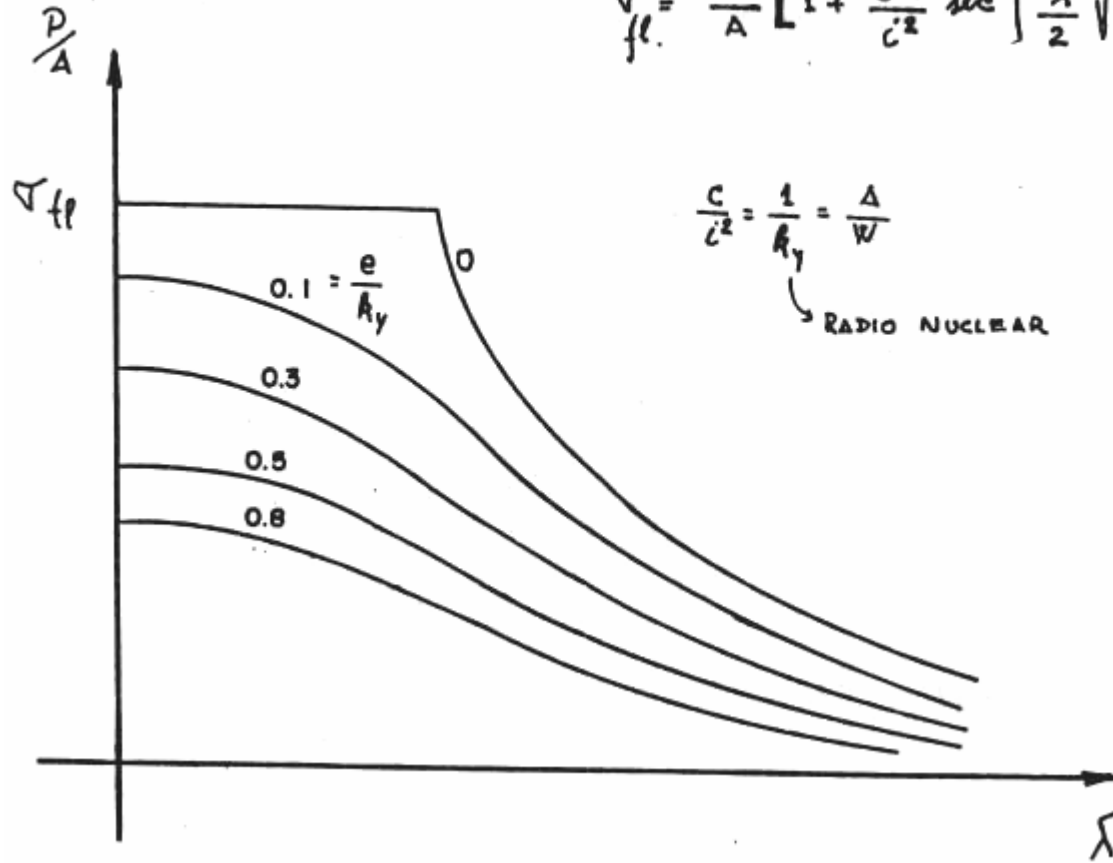
$$\sigma_{max} = \frac{P}{A} + \frac{M_{max} c}{I}$$

$$i = \sqrt{\frac{I}{A}}$$

$$\frac{P_{crit}}{A} = \frac{\sigma_{cl}}{1 + \frac{e c}{i^2} \sec \left(\frac{l}{2i} \sqrt{\frac{P}{AE}} \right)}$$

$$\frac{nP}{A} = \frac{\sigma_{cl}}{1 + \frac{e c}{i^2} \sec \left(\frac{l}{2i} \sqrt{\frac{nP}{AE}} \right)}$$

$$\sigma_{fl} = \frac{P}{A} \left[1 + \frac{e e}{c^2} \sec \left(\frac{\lambda}{2} \sqrt{\frac{P}{A E}} \right) \right]$$



$$\frac{c}{c^2} = \frac{1}{A_y} = \frac{\Delta}{W}$$

RADIO NUCLEAR

$$\frac{\sigma_{fl}}{r} = \frac{P}{A} + \frac{Pe}{W} \sec \left(\frac{\lambda}{2} \sqrt{\frac{n P'}{A E}} \right)$$

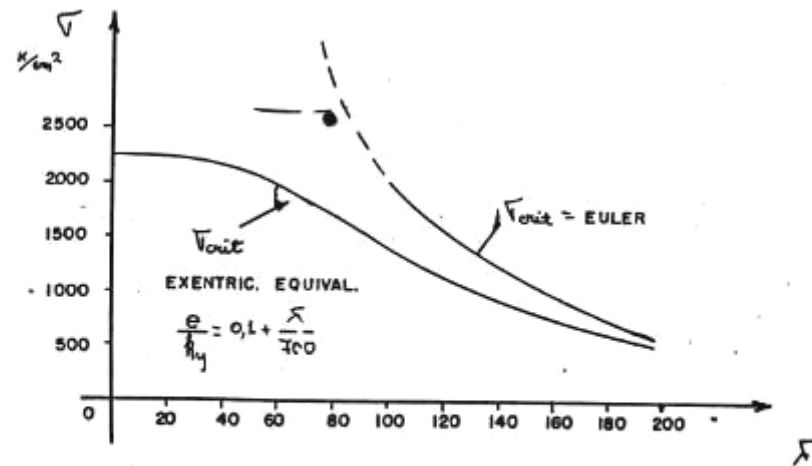
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FORMULA PARA DIMENSIONAMIENTO

$P_{critico}$  depende

- IMPERFECCIONES
- RESISTENCIA
- ESTABILIDAD

# EXCENTRICIDAD EQUIVALENTE



$$\frac{eC}{I^2} = \frac{e}{h_y} = 0.25 \quad \text{P/ COLUMNAS ARTICULADAS EN LOS EXTREMOS}$$

$$\frac{e}{l} = \frac{1}{400} \quad ; \quad (\text{KAYSER}) \quad \text{RECOMENDARLE} \quad \text{P/ TIMOSHENKO}$$

$$\frac{e}{h_y} = 0.06 ; 0.07 \quad (\text{MARSTON ; DENSEN})$$

$$\frac{e}{h_y} = 0.15 \rightarrow 0.6 \quad (\text{MONORIEF})$$

$$\frac{e}{l} = 0.001 \quad (\text{SALMON})$$

$$\frac{e}{h_y} = 0.1 + \frac{\lambda}{700} \quad (\text{PRICHARD})$$

$$\frac{e}{h_y} = 0.1 + \frac{\lambda}{1000} \quad (\text{BASQUIN})$$

$$e = \frac{l}{20} + \frac{l}{500} \quad (\text{DIN 4114})$$

P/ PERFILES DE ALMA LLENA

$$e = \frac{l}{750} + \frac{h}{40}$$

P/PERFILES COMPUESTOS

$$e = \frac{l}{750} + \frac{h}{40} + \frac{h}{160}$$

h = DIMENSION  
DE LA  
SECCION EN EL  
PLANO DEL  
PANDEO



## FÓRMULAS EMPÍRICAS

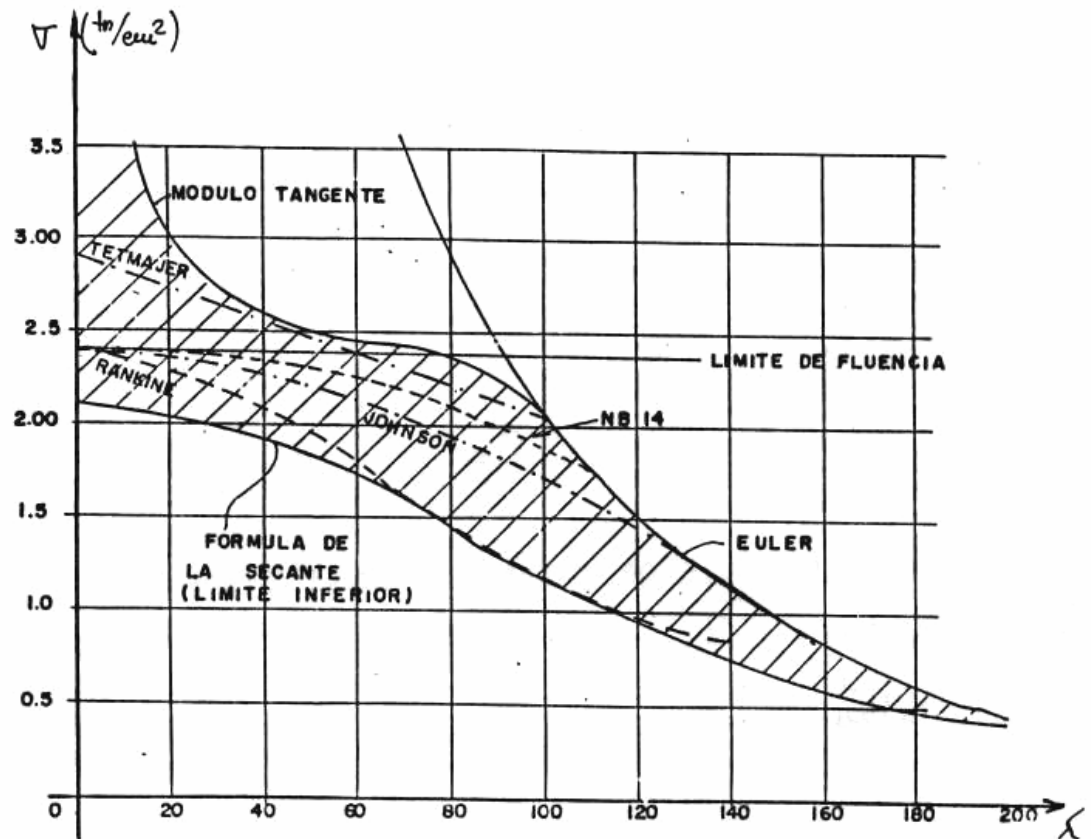
- LINEA RECTA :  $\tau_{crit} = \tau_0 - \alpha \lambda$

- GORDON RANKINE :  $\tau_{crit} = \frac{\tau_0}{1 + (\beta \lambda)^2}$

- PARABOLICA (JOHNSON) :  $\tau_{crit} = \tau_0 - c \lambda^2$

- TETMAJER :  $\tau_{crit} = \tau_0 - a \lambda + b \lambda^2$

ACERO DULCE :  $\tau_f = 2.4 \text{ tn/cm}^2$   
 $\tau_p = 1.9 \text{ tn/cm}^2$



## FORMULAS USUALES

### CHICAGO BUILDING CODE - P/ACERO

|                |                                                   |                    |
|----------------|---------------------------------------------------|--------------------|
| $30 < l < 120$ | $\bar{\sigma}_{crit} = 1125 - 4,921 l$            | LINEA RECTA        |
| $l < 30$       | $\bar{\sigma}_{crit} = 984 \frac{l}{\text{cm}^2}$ |                    |
| $l > 120$      | $\bar{\sigma}_{crit} = \frac{\pi^2 E}{l^2 \eta}$  | $\eta = 2,7$ EULER |

### COLUMN RESEARCH COUNCIL

|               |                                                                |                                                                         |
|---------------|----------------------------------------------------------------|-------------------------------------------------------------------------|
| $l < l_{lim}$ | $\frac{\sigma_{crit}}{\sigma_f} = 1 - \frac{l^2}{2 l_{lim}^2}$ |                                                                         |
|               | $\frac{1/D}{D_f} = \frac{1 - \frac{l^2}{2 l_{lim}^2}}{\eta_1}$ | $\eta_1 = \frac{5}{3} + \frac{3l}{8 l_{lim}} - \frac{l^3}{6 l_{lim}^3}$ |
| $l > l_{lim}$ | $\frac{\sigma_{crit}}{\sigma_f} = \frac{l_{lim}^2}{2 l^2}$     |                                                                         |
|               | $\frac{1/D}{D_f} = \frac{l_{lim}^2}{2 \eta_2 l^2}$             | $\eta_2 = \frac{25}{12} \approx 1,92$                                   |

### AMERICAN INSTITUTE OF STEEL CONSTRUCTION: (AISC)

|           |                                                             |                  |
|-----------|-------------------------------------------------------------|------------------|
| $l < 120$ | $\bar{\sigma}_{crit} = 1200 - 0,0341 l^2$                   | (PARABOLICA)     |
| $l > 120$ | $\bar{\sigma}_{crit} = \frac{1265}{1 + \frac{l^2}{18.000}}$ | (GORDON RANKINE) |

NEW YORK BUILDING CODE

P/ACERO ESTRUCTURAL

$\lambda < 60 \quad \bar{F} = 1055$

$\lambda > 60 \quad \bar{F}_{crit} = \frac{1265}{1 + \frac{\lambda^2}{18.000}} \quad (\text{GORDON RANKINE})$

NB - 14

P/ACERO ESTRUCTURAL A36

$\lambda = 0 \quad \bar{F} = 1200 \frac{\text{K}}{\text{cm}^2}$

$0 < \lambda < 105 \quad \bar{F}_{crit} = 1200 - 0,023 \lambda^2 \quad \text{PARABOLICA}$

$\lambda > 105 \quad \bar{F}_{crit} = \frac{10.363.000}{\lambda^2} \quad (\text{EULER} - \eta = 2)$

P/ACERO  $\bar{F}_{fe} = 3500 \frac{\text{K}}{\text{cm}^2}$

$\lambda = 0 \quad \bar{F} = 1750 \frac{\text{K}}{\text{cm}^2}$

$0 < \lambda < 86 \quad \bar{F}_{crit} = 1750 - 0,0473 \lambda^2 \quad \text{PARABOLICA}$

$\lambda > 86 \quad \bar{F}_{crit} = \frac{10.363.000}{\lambda^2} \quad \text{EULER}$

NB - 11

MADERA

$\lambda < 40 \quad \bar{F}_{crit} = \bar{F}_c$

$40 < \lambda < \lambda_{lim} \quad \bar{F}_{crit} = \bar{F}_c \left( 1 - \frac{1}{3} \frac{\lambda - 40}{\lambda_{lim} - 40} \right)$

$\lambda > \lambda_{lim} \quad \bar{F}_{crit} = \frac{\bar{F}_c^2}{\lambda^2} = \frac{2}{3} \bar{F}_c \left( \frac{\lambda_{lim}}{\lambda} \right)^2 \quad \text{EULER ; } \eta = 4$

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# Final del Programa de Mecánica de Materiales I

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Fin